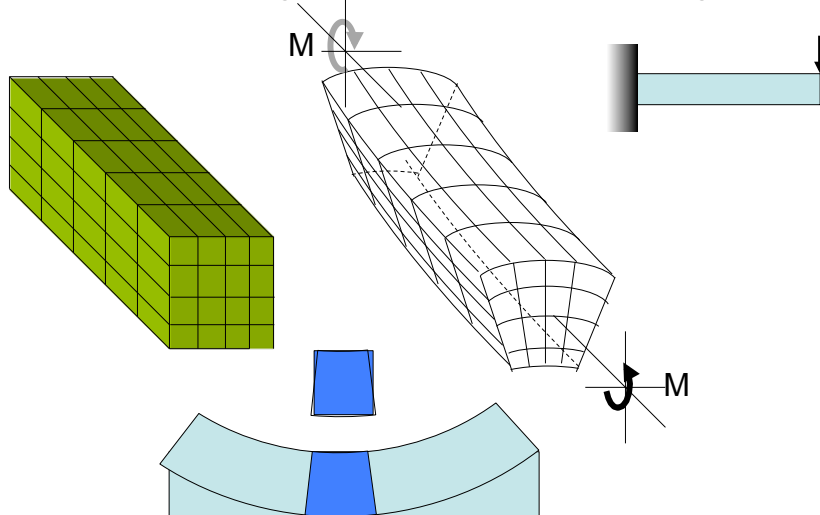


ME222: BEAM

Patrick Kwon
Department of Mechanical Engineering
Michigan State University
East Lansing, Michigan

4.1 Introduction

Pure bending vs. Non-uniform bending



4.2 Bending Deformation

Before Bending,
 $L_{ab} = L_{ef} = \rho d\theta$

After Bending,
 $L_{ab} = dx = \rho d\theta$
 $L_{ef} = (\rho - y)d\theta$

$\Delta L_{ef} = dx - (\rho - y)d\theta = yd\theta$

$\epsilon_x = \frac{\Delta L_{ef}}{L_{ef}} = -\frac{yd\theta}{\rho d\theta} = -\frac{y}{\rho}$

$\sigma_x = E\epsilon_x = -E\frac{y}{\rho} = -\left(\frac{y}{c}\right)\sigma_{\max}$

4.3 Flexure Formula

- No resultant force on the surface: Neutral Plane (N.P.)

$$\sum F_x = 0; \quad \int_A \sigma_x dA = -\frac{\sigma_{\max}}{c} \int_A y dA = 0 \quad \text{First Moment of An Area}$$

- The integral of the elemental moment over the surface must equal to the bending moment

$$\sum M = 0; \quad M = -\int_A \sigma_x y dA = -\frac{E}{\rho} \int_A y^2 dA$$

- Moment of Inertia: $I = \int_A y^2 dA$
(Second Moment of Area)

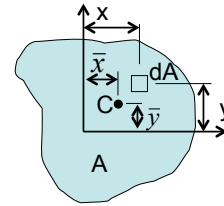
- Flexure Formula: $\sigma_x = -\frac{My}{I} \left(= -\frac{Ey}{\rho} = -\frac{\sigma_{\max} y}{c} \right)$

Moments of Areas

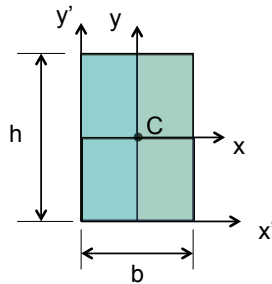
- First Moment of An Area: Centroid of An Area

First Moment of Area A w.r.t. x-axis: $Q_x = \int_A y dA = A\bar{y}$

First Moment of Area A w.r.t. y-axis: $Q_y = \int_A x dA = A\bar{x}$



Centroid of Area A: C (\bar{x}, \bar{y}) at which Q_x and Q_y will be zeros.



Example: Rectangle

$$Q_x = A\bar{y} = (bh)(0) = 0 \quad Q_{x'} = A\bar{y}' = (bh)\frac{h}{2} = \frac{bh^2}{2}$$

$$Q_y = A\bar{x} = (bh)(0) = 0 \quad Q_{y'} = A\bar{x}' = (bh)\frac{b}{2} = \frac{b^2h}{2}$$

First Moment of An Area

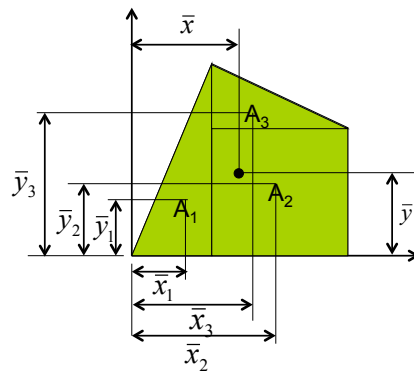
$$Q_x = \int_A y dA = \sum_{i=1}^n A_i \bar{y}_i$$

$$Q_y = \int_A x dA = \sum_{i=1}^n A_i \bar{x}_i$$

Centroid (\bar{x}, \bar{y})

$$\bar{x} = \frac{\sum_{i=1}^n A_i \bar{x}_i}{\sum_{i=1}^n A_i}$$

$$\bar{y} = \frac{\sum_{i=1}^n A_i \bar{y}_i}{\sum_{i=1}^n A_i}$$



Second Moment (Moment of Inertia)

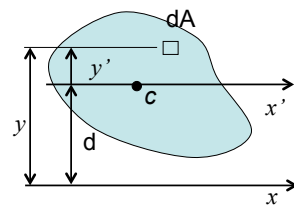
Second Moment w.r.t. x-axis: $I_x = \int_A y^2 dA$

Second Moment w.r.t. y-axis: $I_y = \int_A x^2 dA$

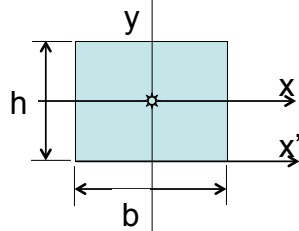
Polar Moment of Inertia: $J = \int_A \rho^2 dA = I_x + I_y$

Parallel-Axis Theorem

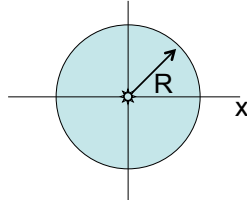
$$\begin{aligned} I_x &= \int_A y^2 dA = \int_A (y' + d)^2 dA \\ &= \int_A y'^2 dA + 2d \int_A y' dA + d^2 \int_A dA \\ &= \bar{I}_{x'} + 2dQ_{x'} + Ad^2 = \bar{I}_{x'} + Ad^2 \\ \therefore Q_{x'} &= A\bar{y}' = A(0) = 0 \end{aligned}$$



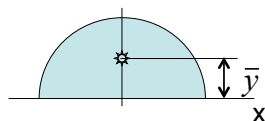
Moment of Inertia



$$\begin{aligned} I_x &= \int_A y^2 dA = \int_{-b/2}^{b/2} \int_{-h/2}^{h/2} y^2 dy dx \\ &= \int_{-b/2}^{b/2} \left(\frac{y^3}{3} \Big|_{-h/2}^{h/2} \right) dx = \int_{-b/2}^{b/2} \frac{h^3}{12} dx = \frac{bh^3}{12} \\ I_{x'} &= \frac{bh^3}{12} + bh \left(\frac{h}{2} \right)^2 = \frac{bh^3}{3} \end{aligned}$$



$$\begin{aligned} I_x = I_y &= \int_A y^2 dA = \int_0^R \int_0^{2\pi} r^2 \cos^2 \theta dr d\theta \\ &= \int_0^R \left(\frac{\theta}{2} + \frac{\sin 2\theta}{4} \Big|_0^{2\pi} \right) r^3 dx = \int_0^R \pi r^3 dx = \frac{\pi R^4}{4} \\ J_0 = I_x + I_y &= \frac{\pi R^4}{4} + \frac{\pi R^4}{4} = \frac{\pi R^4}{2} \end{aligned}$$



$$I_x = I_y = \frac{\pi r^4}{8} \quad \bar{y} = \frac{4r}{3\pi}$$