

# **Nonlinear Dynamics of Longitudinal Ground Vehicle Traction**

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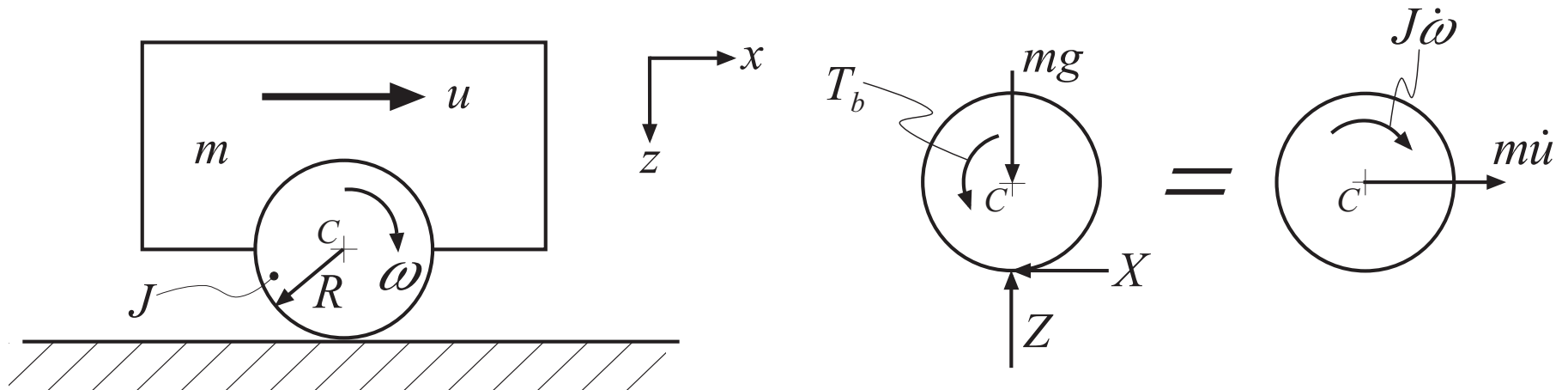
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# Outline

- Introduction
- Single Wheel Models
  - Braking (SWBM)
  - Acceleration (SWAM)
- Two Wheel Model
  - Braking (2WBM)
- Conclusions and Directions for Future Work

# The Single Wheel Braking Model



Dynamics are governed by

$$\begin{aligned} Z &= mg \\ m\dot{u} &= -X \\ J\dot{\omega} &= RX - T_b \end{aligned}$$

# The Tire/Road Interface

- Need an expression relating  $X$  and  $Z$  ●
- 

Friction Law (Creep Force Equation)

$$X = \mu(s)Z$$

$\mu$  longitudinal force coefficient

$s$  longitudinal wheel slip

# Longitudinal Wheel Slip

- Measure of the difference between vehicle speed  $u$  and rolling speed  $\omega R$  of wheel

$$s \equiv \frac{u - \omega R}{\max(u, \omega R)}, \quad \begin{cases} \omega R \leq u & \text{(braking)} \\ u \leq \omega R & \text{(acceleration)} \end{cases}$$

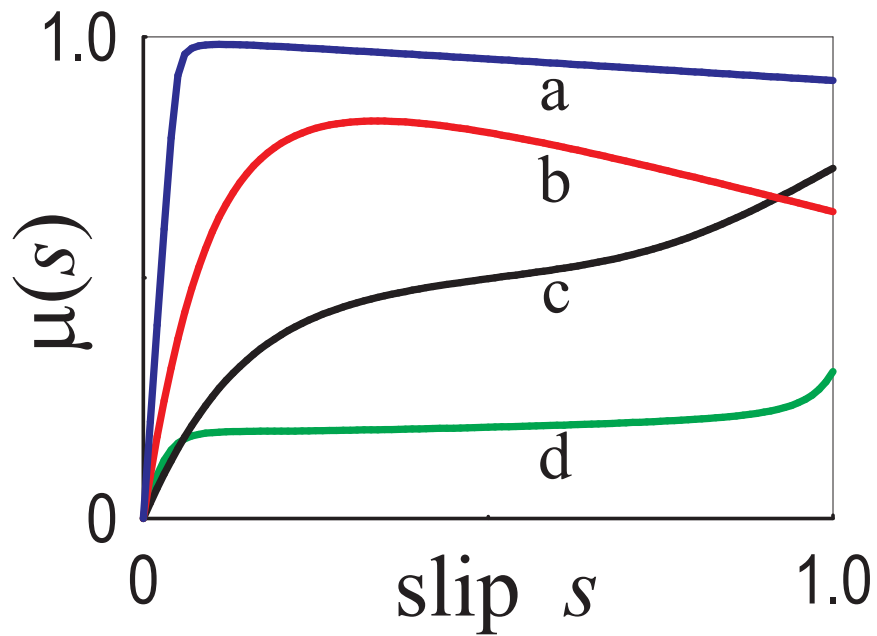
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$$s = -1 \quad (u = 0) \quad \Rightarrow \quad \text{pure slip}$$

$$s = 0 \quad (u = \omega R) \quad \Rightarrow \quad \text{no slip, or free rolling}$$

$$s = 1 \quad (\omega R = 0) \quad \Rightarrow \quad \text{wheel lockup}$$

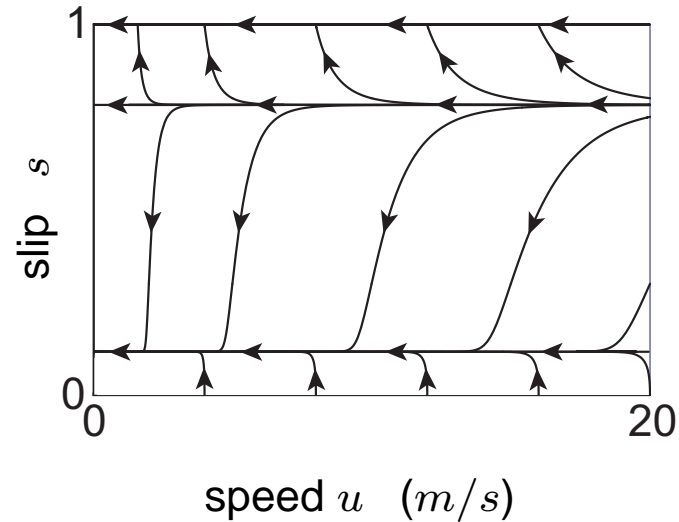
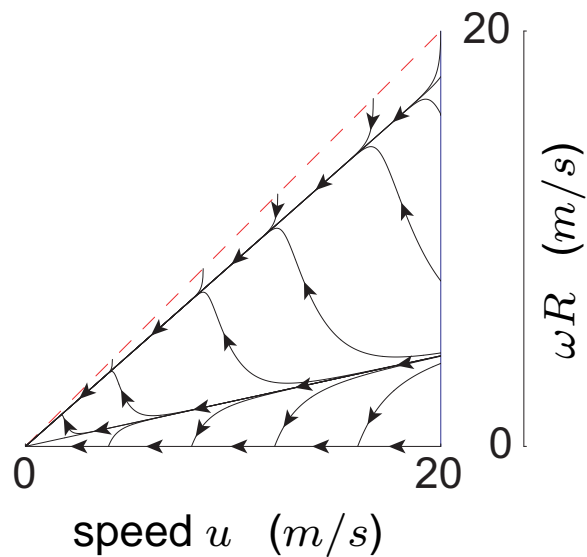
# Friction Characteristics



- a)* Dry asphalt
- b)* Wet asphalt
- c)* Gravel
- d)* Packed Snow

# Choice of Dynamic States

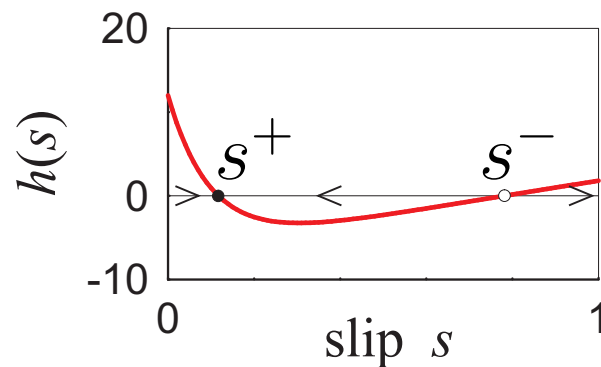
$(u, \omega R)$  vs.  $(u, s)$  state space



# Equations of Motion

$$\left. \begin{aligned} \dot{u} &= -\mu(s)g \\ \dot{s} &= \frac{g}{u}h_b(s) \end{aligned} \right\} \quad u > 0, \quad s \in I$$

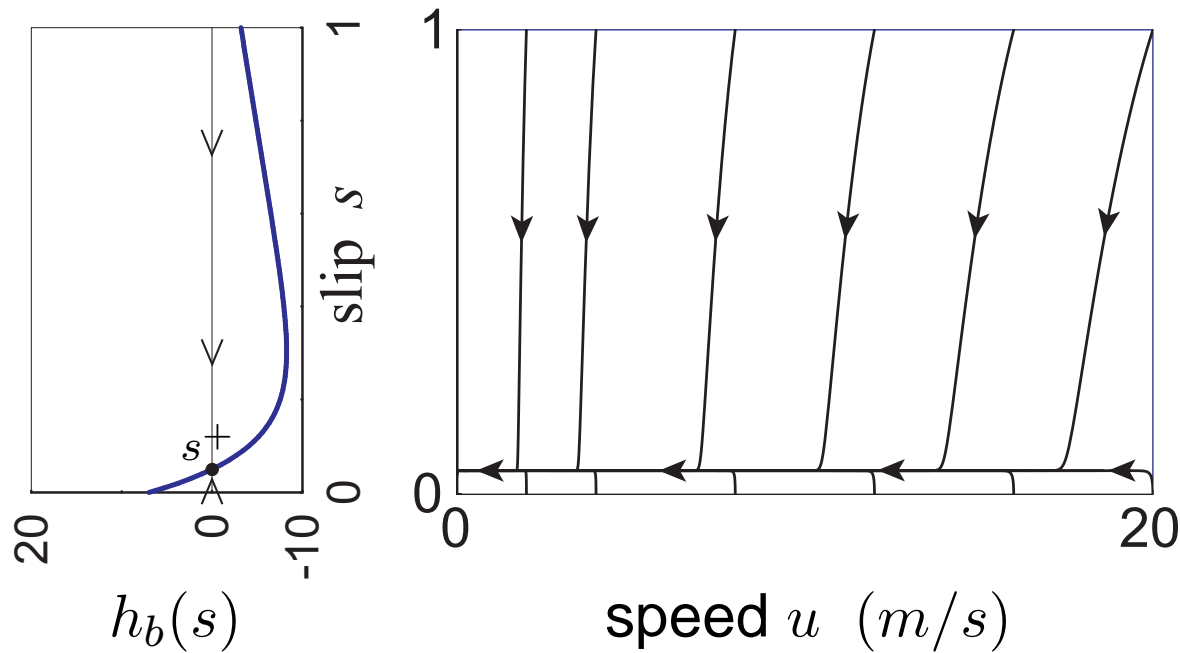
$$h_b(s) = \mu(s)(s - 1 - \Psi) + \Upsilon_b$$



$$\Upsilon_b = 12$$

$$\Psi = 15$$

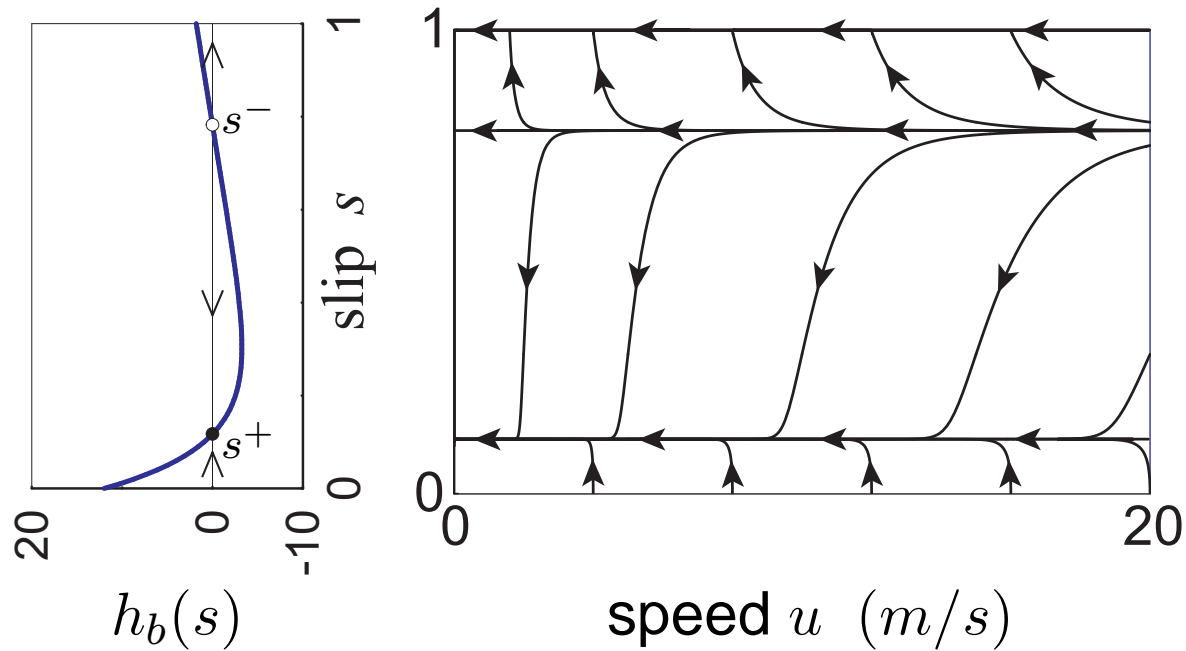
# State Space Description



$$\Upsilon_b = 7$$

**Stable Braking**

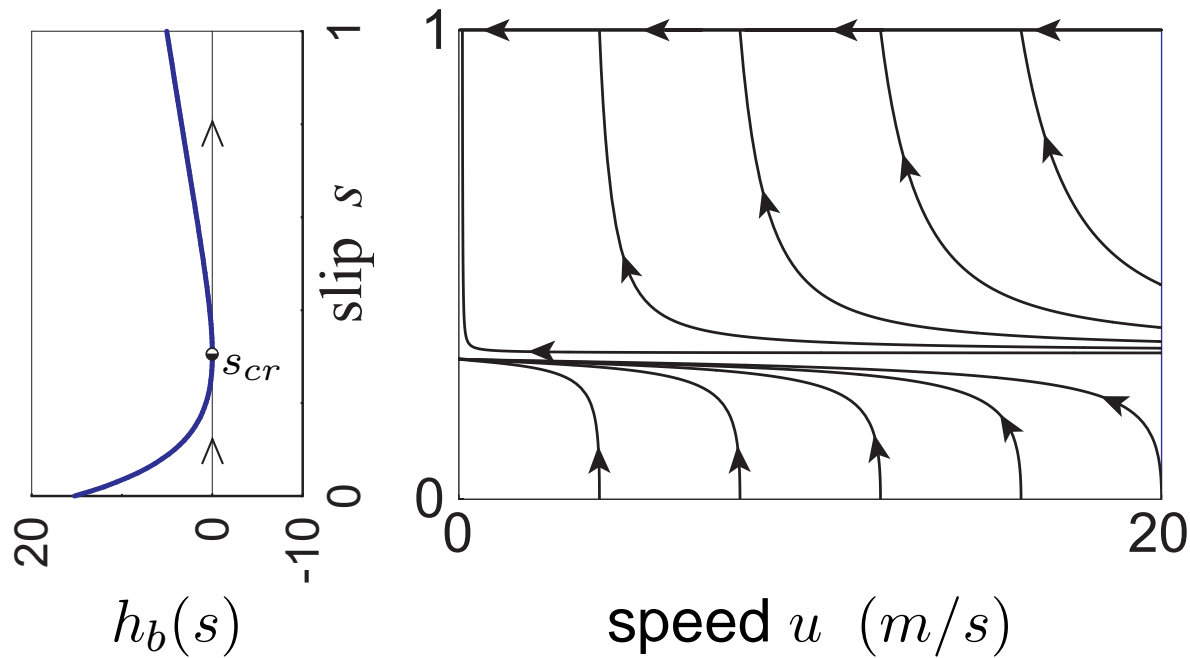
# State Space Description



$$\Upsilon_b = 12$$

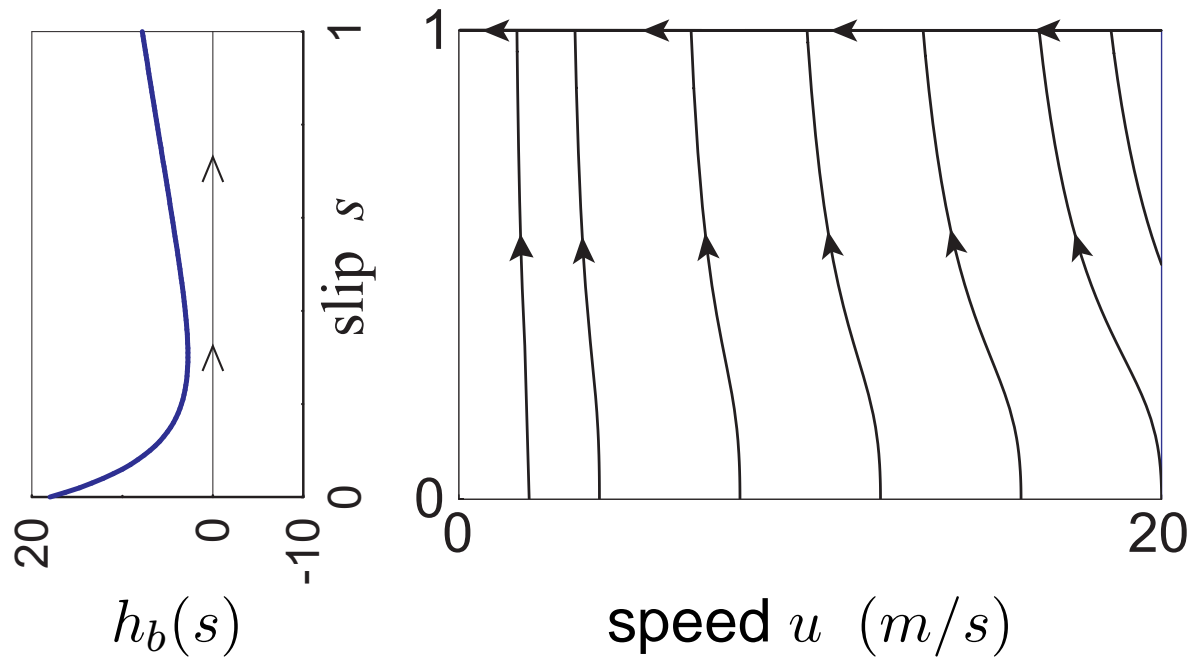
**Mixed Braking**

# State Space Description



$$\Upsilon_b = \Upsilon_{b_{cr}} = 15.250 \quad \text{Unstable Braking}$$

# State Space Description



$$\Upsilon_b = 18$$

**Unstable Braking**

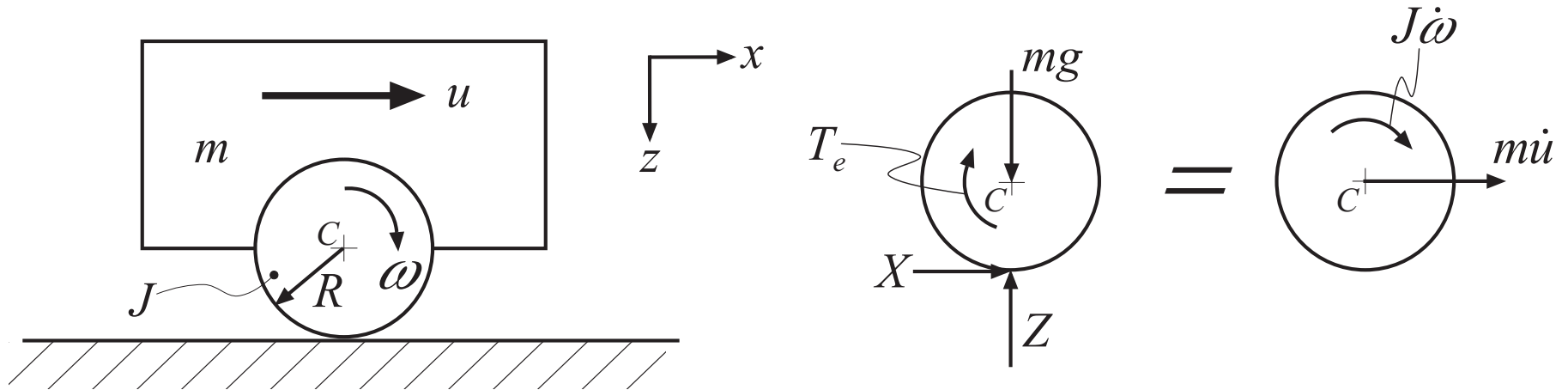
# The Transition to Unstable Braking

- Maximum brake torque:

$$T_{b_p} = \begin{cases} mgR\mu(s_p) & \text{(assumed)} \\ mgR\mu(s_{cr}) \left[ 1 + \frac{1}{\Psi}(1 - s_{cr}) \right] & \text{(actual)} \end{cases}$$

- Fundamental assumptions:
  1.  $\Psi \gg 1$ .
  2.  $s_p$  can be reached.

# The Single Wheel Acceleration Model

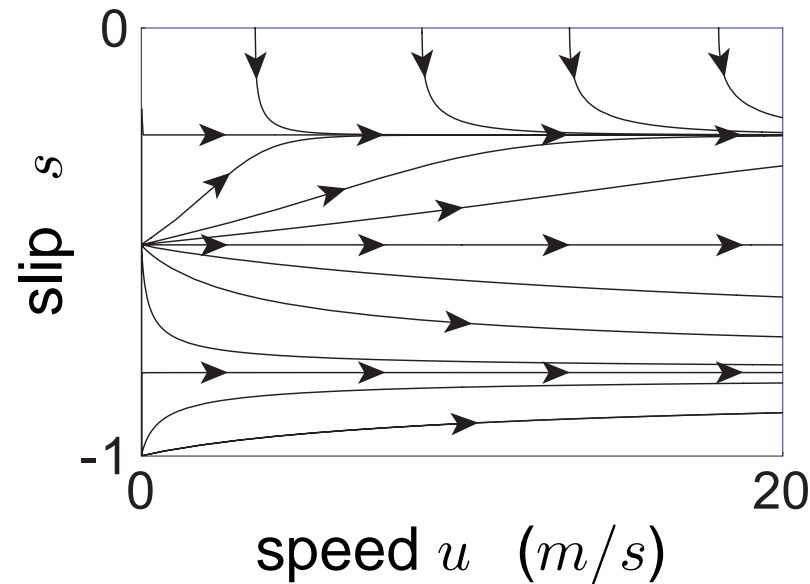
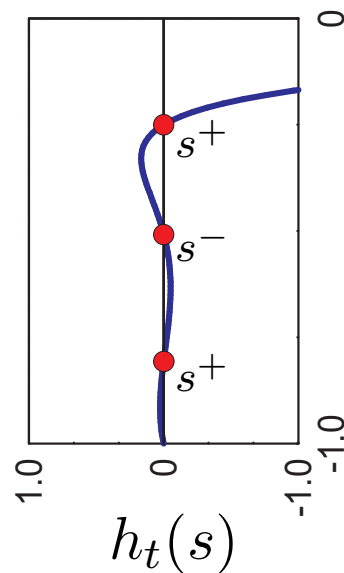


Equations of Motion:

$$\left. \begin{aligned} \dot{u} &= \mu(s)g \\ \dot{s} &= \frac{g}{u} h_t(s) \end{aligned} \right\}, \quad u > 0, \quad s \in [-1, 0].$$

# Example State Space

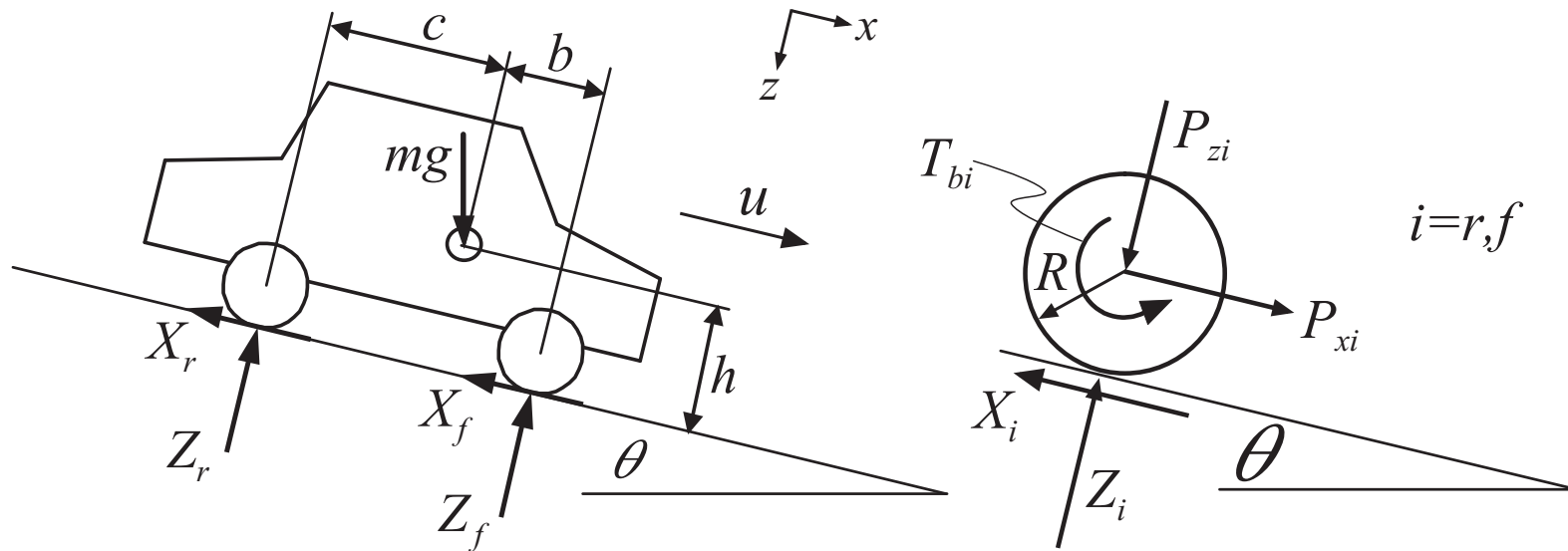
$$h_t(s) = (s + 1)^2 \left[ (s + 1)^{-1} \mu(s) + \Psi \mu(s) - \Upsilon_e \right]$$



$$\Upsilon_e = 15.65$$

**Mixed Acceleration**

# The Two Wheel Braking Model



$$X_i = \mu(s_i)Z_i, \quad s_i \equiv \frac{u - \omega_i R}{u}, \quad i = r, f$$

$$Z_r = mg \left( \frac{b}{l} \cos \theta - \frac{h}{l} \sin \theta \right) + m\dot{u} \frac{h}{l}$$

$$Z_f = mg \left( \frac{c}{l} \cos \theta + \frac{h}{l} \sin \theta \right) - m\dot{u} \frac{h}{l}$$

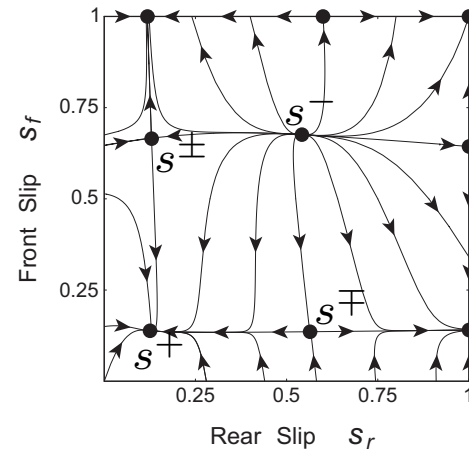
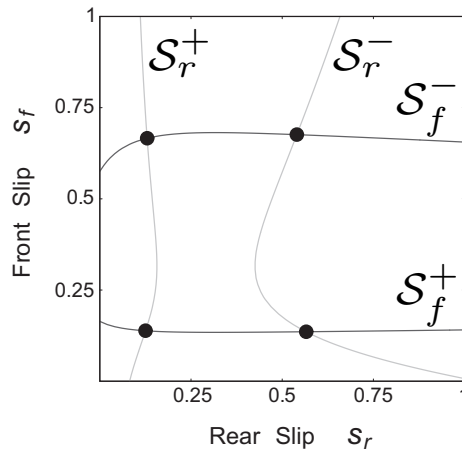
# Equations of Motion

$$\left. \begin{aligned} \dot{u} &= -g (\Lambda_b(\mathbf{s}) \cos \theta - \sin \theta) \\ \dot{s}_r &= \frac{g}{u} h_{br}(\mathbf{s}) \\ \dot{s}_f &= \frac{g}{u} h_{bf}(\mathbf{s}) \end{aligned} \right\}, \quad u > 0, \mathbf{s} \in I \times I.$$

$$\mathbf{s} = (s_r, s_f)$$

$$h_{bi}(\mathbf{s}) = (s_i - 1) (\Lambda_b(\mathbf{s}) \cos \theta - \sin \theta) - \mu(s_i) \Psi \lambda_i(\mathbf{s}) + \Upsilon_{bi},$$
$$i = r, f$$

# Steady Slip Conditions



$$\mathcal{S}_i^\pm = \{s \mid h_{bi}(s) = 0, h'_{bi}(s) \leq 0\}, \quad i = r, f.$$

$$s^+ = \mathcal{S}_r^+ \cap \mathcal{S}_f^+$$

Stable Node

$$s^- = \mathcal{S}_r^- \cap \mathcal{S}_f^-$$

Unstable Node

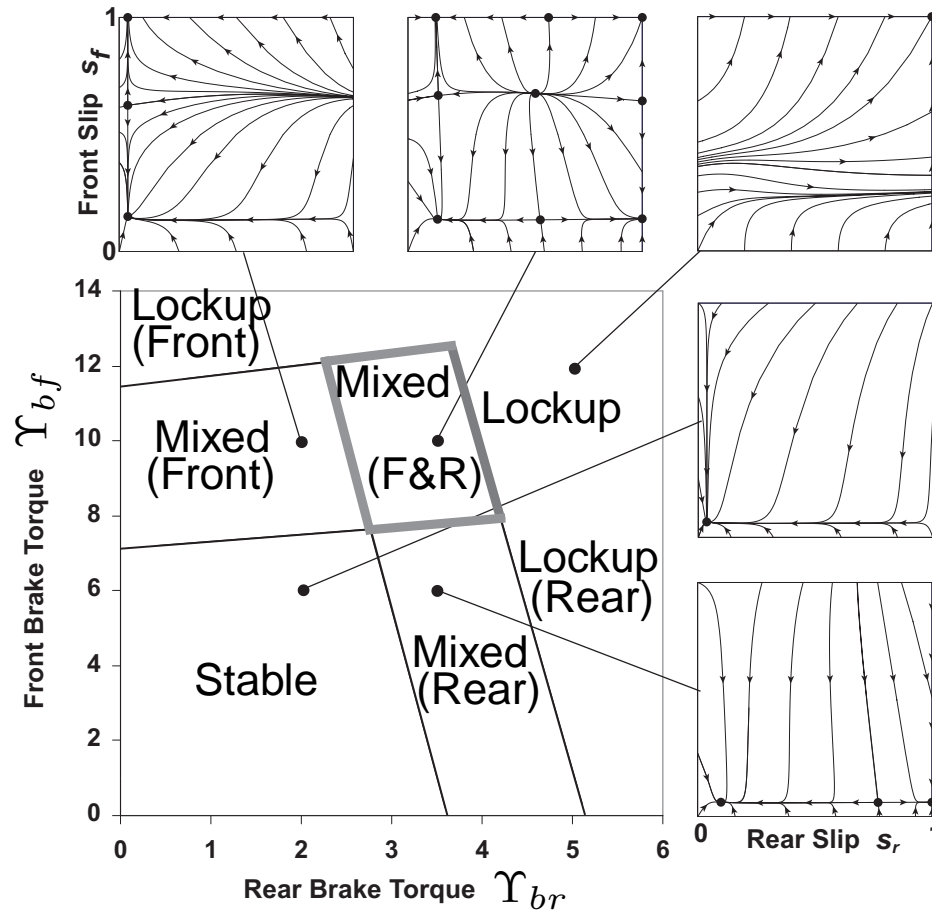
$$s^\pm = \mathcal{S}_r^+ \cap \mathcal{S}_f^-$$

Saddle

$$s^\mp = \mathcal{S}_r^- \cap \mathcal{S}_f^+$$

Saddle

# State Space Description



# Directions for Future Work

- More detailed parameter studies
- Use of the two-wheel braking model to assess brake proportioning strategies
- A more thorough study of the two wheel acceleration model
- The effects of cornering
- the incorporation of these models into ABS/TCS development

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